



A Compensator Based Control for Non-Interacting Tank System

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ABSTRACT:Proportional-Integral-Derivative (PID) controllers are commonly employed in process control applications to obtain the desired response. They are found to yield optimum results in such applications. An attempt has been made in this article to employ a Non-Iterative compensator as an alternative to conventional controller and intelligent controller for a two tank Non-Interacting system. Experimental results are obtained by observing the transient response of liquid level using the compensator as well as Conventional and Intelligent controller. The compensator based control is found to yield better results than a PID controller, Internal Model Control (IMC) based PID controller and Fuzzy tuning PID. Since the implementation of the Non-Iterative compensator involves only three passive components, it can be extended appropriately even for interacting tanks system and many such processes.

KEYWORDS:Non-Interacting Tank system, Ziegler Nichols tuned PID controller, IMC based PID controller, Fuzzy tuning PID and Non-Iterative Compensator.

I.INTRODUCTION

The control of liquid level in Non-Interacting tank system is a major problem in process industries like chemical industries, paper & pulp industries, pharmaceutical industries and power plants. The liquid in the tank will be processed by chemical or mixing treatment, but often the level of liquid in the tank must be controlled and maintained [1]. Based on the accuracy of the levels, the quality of the final product is determined [2]. The main objective of this work is to maintain and control the level of Non-Interacting tank at desired setpoint.

In proposed work, level of a non-interacting tank is controlled by conventional PID tuning techniques, PID tuning using intelligent controller technique and Non-Iterative compensator technique. Result obtained from these techniques are compared and analyzed for the better improvement of the system to obtain its desired setpoint with less time consumption.

Generally PID controller is mainly used in all process industries to control the certain process. Because of their performance, it is said to be work-horse of process industries [3]. Different tuning methods are implemented to find the effective controller parameters, in order to obtain better results [4]. In this work PID parameters are tuned using Ziegler-Nichols (Z-N), IMC based PID and Fuzzy tuning PID. Even though tuning of PID parameters obtains better results there are some difficulties to tune the PID parameters [5]. To overcome the problems of tuning methods a simple Non-Iterative Compensator technique is proposed in this work to control the level of Non-Interacting tank.

Nowadays, design of compensator using lead and lag network for control system application has become a common practice to obtain the desired steady state and transient response with safe stability margins [6]. It is observed that such compensators are well suited for the systems which have only real poles [7]. To meet the desired specifications of the Non-Interacting level process, the modified methodology is proposed in this paper.

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II.MATHEMATICAL MODELING

Fig 1 shows the model of Non-interacting tank system.
Let,

q_i = Input flow of first tank

q_o = Output flow of first tank & Input flow of Second

q_{io} = Output flow of second tank

h_a = Height of the liquid in first tank

h_b = Height of the liquid in second tank

R_a = Restriction element of first tank

R_b = Restriction element of second tank

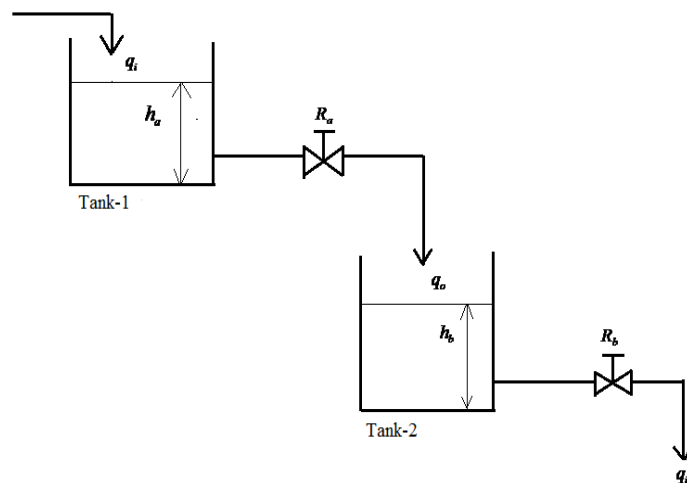


Fig. 1 General model of Non-Interacting tank System

Any changes in level h_a of Tank-1 will change the level h_b of Tank-2, but the converse is not possible. This type of system is said to be Non-Interacting tank system [8].

According to the Law of conservation of Mass,
Tank-1:

$$q_1 - q_2 = C_1 \frac{dh_1}{dt} \quad (1)$$

q_1 = Input flow to the first tank

q_2 = Output flow from the first tank & Input flow to the second tank

C_1 = Accumulation of Tank-1



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h_1 = Height of the Tank-1

Since the Resistance of the tank-1 is small,

q_2 gets negligible,

So equation (1) becomes,

$$q_1 = C_1 \frac{dh_1}{dt} \quad (2)$$

Tank-2:

$$q_2 - q_3 = C_2 \frac{dh_2}{dt} \quad (3)$$

q_2 - Input Flow to the second Tank

q_3 - Output flow from the second tank

C_2 - Accumulation of tank-2

h_2 - Height of the second tank

Consider the flow rate of q_1 as the flow rate of q_2 , since the valve coefficient of q_2 is negligible.

Valve coefficient for q_3 is,

$$q_3 = \frac{h_2}{R_2} \quad (4)$$

Therefore, the overall Transfer function of the system is,

$$\frac{H_2(S)}{Q_1(S)} = \frac{R_2}{R_2 C_1 C_2 S^2 + C_1 S} \quad (5)$$

By experimental method, the values for the above Transfer function are calculated.

Therefore the Non- Interacting Tank model is,

$$\frac{H_2(S)}{Q_1(S)} = \frac{0.1}{2S^2 + S} \quad (6)$$

III.ZIEGLER- NICHOLS TUNING METHOD

PID parameters are obtained from the Ziegler–Nichols tuning formula shown in Table I. By adjusting the gain of the system, critical gain (K_{cr}) can be measured and oscillation period also measured in terms of critical period (P_{cr}). The PID values are tuned from oscillatory response parameters [9]. Increase K_p from 0 to critical value K_{cr} at which the output first exhibits sustained oscillations [10].



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Table I Z-N method for PID tuning

Type of controller	K_p	T_i	T_d
PID	$0.6 K_{cr}$	$0.5 P_{cr}$	$0.125 P_{cr}$

PID parameters are determined by substituting the values of K_{cr} (critical gain) = 6.7 and P_{cr} (critical period) = 27 seconds in the Table I.

Therefore $K_p = 4$, $K_i = 1$, $K_d = 0.1$

The response of Non-Interacting tank using Z-N tuned PID controller is shown in fig 3 and the time domain specification for the response is shown in Table II. Since trial and error procedure must be performed to get the oscillatory response it is said to be a time consuming method [11]. To overcome the above said drawback, IMC based PID controller is proposed.

Table II Time domain Specifications for Non-Interacting tank using Z-N PID tuning

Type of controller	Settling Time	Peak Overshoot	Rise Time
PID	120 Seconds	67%	3.678 Seconds

IV.IMC BASED PID

PID parameters are found by IMC based PID tuning formula shown in Table III. IMC design procedure could be used to develop an equivalent PID-type control law. For stable process with no time delay, the IMC based PID procedure gives exactly the same feedback performance as IMC [12] [13].

Table III Controller parameters for IMC based PID method

Types of controller	K_p	T_i	T_d
PID	$\frac{(\tau_1 + \tau_2)}{K_c \lambda}$	$\tau_1 + \tau_2$	$\frac{\tau_1 \tau_2}{\tau_1 + \tau_2}$

Where,

τ_1 & τ_2 = Time Constant

K_c = Gain

λ = optimum filter tuning factor (Assumable)

By substituting the values of $\tau_1 = 1$ sec, $\tau_2 = 2$ sec, $K_c = 0.1$ and $\lambda = 1$ in the table 3, PID parameters are determined. Therefore, $K_p = 8$, $K_i = 2.83$, $K_d = 3$

From fig 3 it is observed that the response of Non interacting tank using IMC based PID controller has high peak overshoot and the time taken to obtain desired response are high. The time domain specification for the response is shown in Table IV. In order to overcome the drawback of IMC based PID controller tuning of PID using intelligent controller technique is proposed.



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Table IV Time domain Specifications for Non-Interacting tank using IMC based PID

Type of controller	Settling Time	Peak Overshoot	Rise Time
IMC based PID	109 Seconds	63%	2.68 Seconds

V.FUZZY TUNED PID

In contrast to conventional control techniques, fuzzy logic control (FLC) is best utilized in complex processes that can be controlled by a skilled human operator without much knowledge of their underlying dynamics. The basic idea behind FLC is to incorporate the "expert experience" of a human operator in the design of the controller in controlling a process whose input – output relationship is described by collection of fuzzy control rules (e.g., IF-THEN rules) involving linguistic variables rather than a complicated dynamic model[14][15]. E and $\frac{de}{dt}$ (Error and derivative error)

are the inputs for Fuzzy logic controller. Hence Fuzzy Controller is used to develop the fuzzy rules for rewriting the PID parameters to meet desired specifications. Where, R (T) is the desired input and Y (T) is the output. Fig 2 shows the general block diagram of fuzzy tuning PID for controlling the level of Non-Interacting tank.

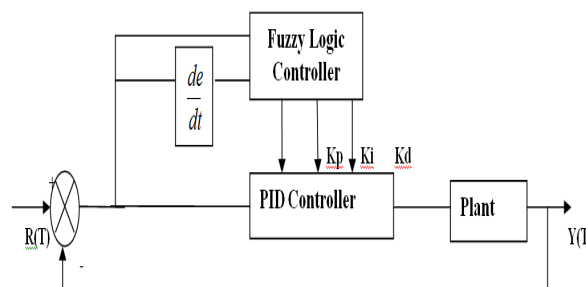


Fig. 2 General Block Diagram of Fuzzy Tuning PID

Based on the inputs i.e., error and change in error, the output response for level is obtained. The triangular shape membership functions are chosen for both the inputs. The membership functions of input and output control signals. By using the input and output membership functions, the fuzzy IF-THEN rules are framed for Non-Interacting system. Table V shows the fuzzy rules for controlling the level of Non-Interacting tank based on the error and change in error of the level.

Table V Fuzzy Rules for level control

E / de/dt	E1	E2	E3	E4
LWF	LL	LL	LL	LL
MWF	ML	ML	LL	LL
HWF	VHL	HL	ML	LL
VHWF	VHL	VHL	HL	ML

From fig 3 it is observed that response of Non-interacting tank using Fuzzy Tuning PID controller removes peak overshoot and less time consumption for output to be settled and its time domain specifications are shown in Table VI. Though intelligent controller removes the peak overshoot and improves in settling time, it requires design experience



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which makes them complicated [16]. The drawback present in the intelligent controller is overcome by the simple Non-Iterative compensator technique in the proposed work.

Table VI Time domain Specifications for Fuzzy Tuning PID

Type of controller	Settling Time	Peak Overshoot	Rise Time
Fuzzy Tuning PID controller	60 Seconds	0%	22.68 Seconds

VI. PROPOSED METHOD

A non-iterative first order compensator design procedure is employed in [17] [18] to meet the desired specifications on phase margin (PM), gain margin (GM) and steady state error (ess).

- The required gain (Kc) of the system is selected to meet the steady state requirement, where $K_c = K_p/A$. K_p = Proportional gain, A= constant >1. Therefore $K_c=2$.
- Transfer function of the compensator to be designed is expressed as,

$$G_{comp}(s) = \frac{(1 + T_a s)}{(1 + T_b s)}$$

Where, T_a and T_b evaluated from equation 7.

$$\left| G_{pc}(j\omega) \right| = K_c \left| \frac{1 + j\omega T_a}{1 + j\omega T_b} \right| \left| G_p(j\omega) \right| = 1 \quad (7)$$

Where, $\left| G_p(j\omega) \right|$ = Gain of the uncompensated system is evaluated at the desired gain crossover frequency ω_g .

Gain of the compensated system transfer function at the desired gain crossover frequency, ω_g is $\left| G_{pc}(j\omega) \right|$.

$\omega_g = 42.7$ rad/sec and $\left| G_p(j\omega) \right|_{42.4}$ (or) gain = 0.78 obtained from the bode plot.

Therefore $T_a = 107.4145$ and $T_b = 260.4167$, the obtained Transfer function of the compensator is,

$$G_c(s) = \frac{(1 + 107.4145 S)}{(1 + 260.4167 S)}$$

Fig 3 clearly shows that response of non-iterative compensator improves in settling time and it obtained zero peak overshoot and its time domain specifications are shown in Table VII.

Table VII Time domain Specifications for Non-Interacting tank using Non-iterative Compensator

Type of controller	Settling Time	Peak Overshoot	Rise Time
Non Iterative Compensator	50 Seconds	0%	19.56 Seconds



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VII. RESULT AND DISCUSSION

From fig 3 & Table VIII it is clearly shown that the Non-Iterative compensator shows better result with less settling time and zero peak overshoot when compared to conventional controller and intelligent controller.

Table VIII Comparison of controllers

Controllers	Ziegler Nichols	IMC based PID	Fuzzy Tuning PID	Non-iterative compensator
Settling time(seconds)	89	70	57	45
Peak overshoot (%)	67	63	0	0
Rise time(Seconds)	3.678	2.68	22.68	19.56

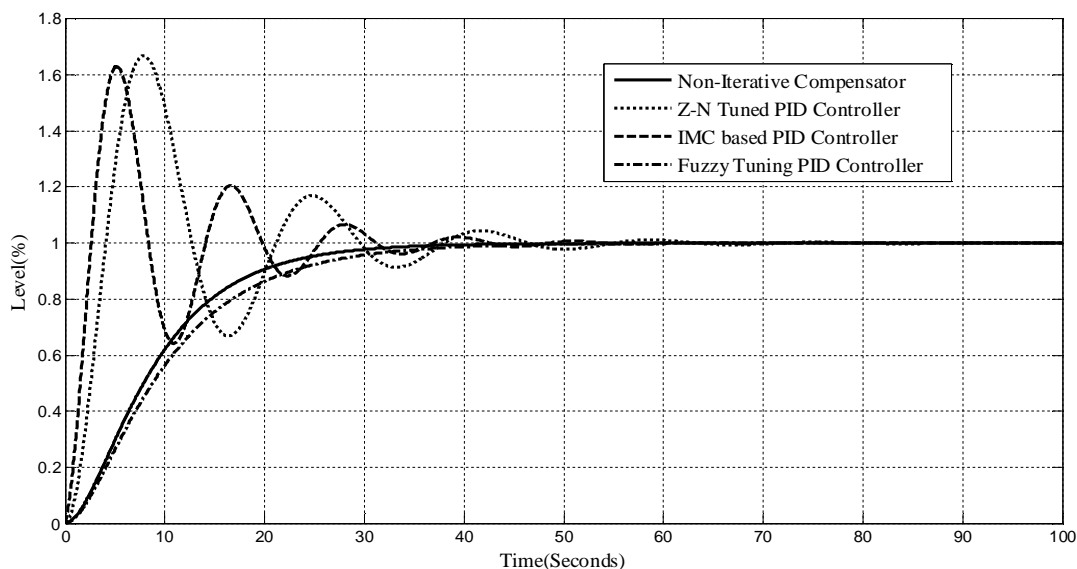


Fig. 3 Response of Non-Interacting tank using various controllers

VIII. CONCLUSION

In this article, an attempt has been made to employ a simple Non-iterative Compensator as an alternative to PID controller for a Non-Interacting tank system. Z-N based PID controller tuning and IMC based PID controller tuning are found to exhibit a very high overshoot whereas fuzzy tuning PID exhibit larger settling time for the desired tank level. However the Non-iterative compensator is found to yield better results than these three tunings since the system settles down faster with zero overshoot. Moreover the compensator can be easily implemented in real time than the PID controller since it involves only three components. The effect of Non linearity such as saturation in PID controller can also be reduced to a greater extent since the gain K_c is very small compared to K_p of PID controller. Hence the compensator can be used as an effective controller to get the desired response in Non-Interacting tank system.



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